

ME 412 – CFD_HT LAB PROJECT

Title – Study of laminar flow and heat transfer in a square channel with 30 deg inline angled baffle turbulators using OpenFOAM.

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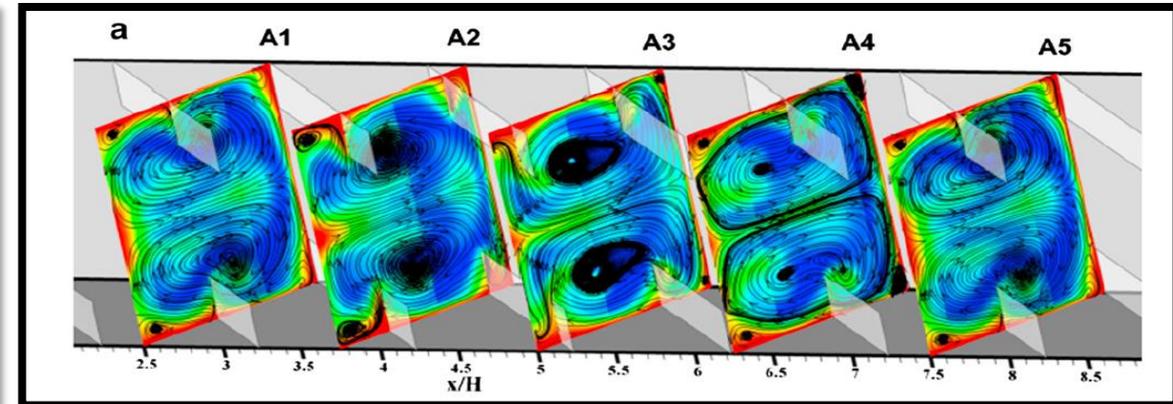
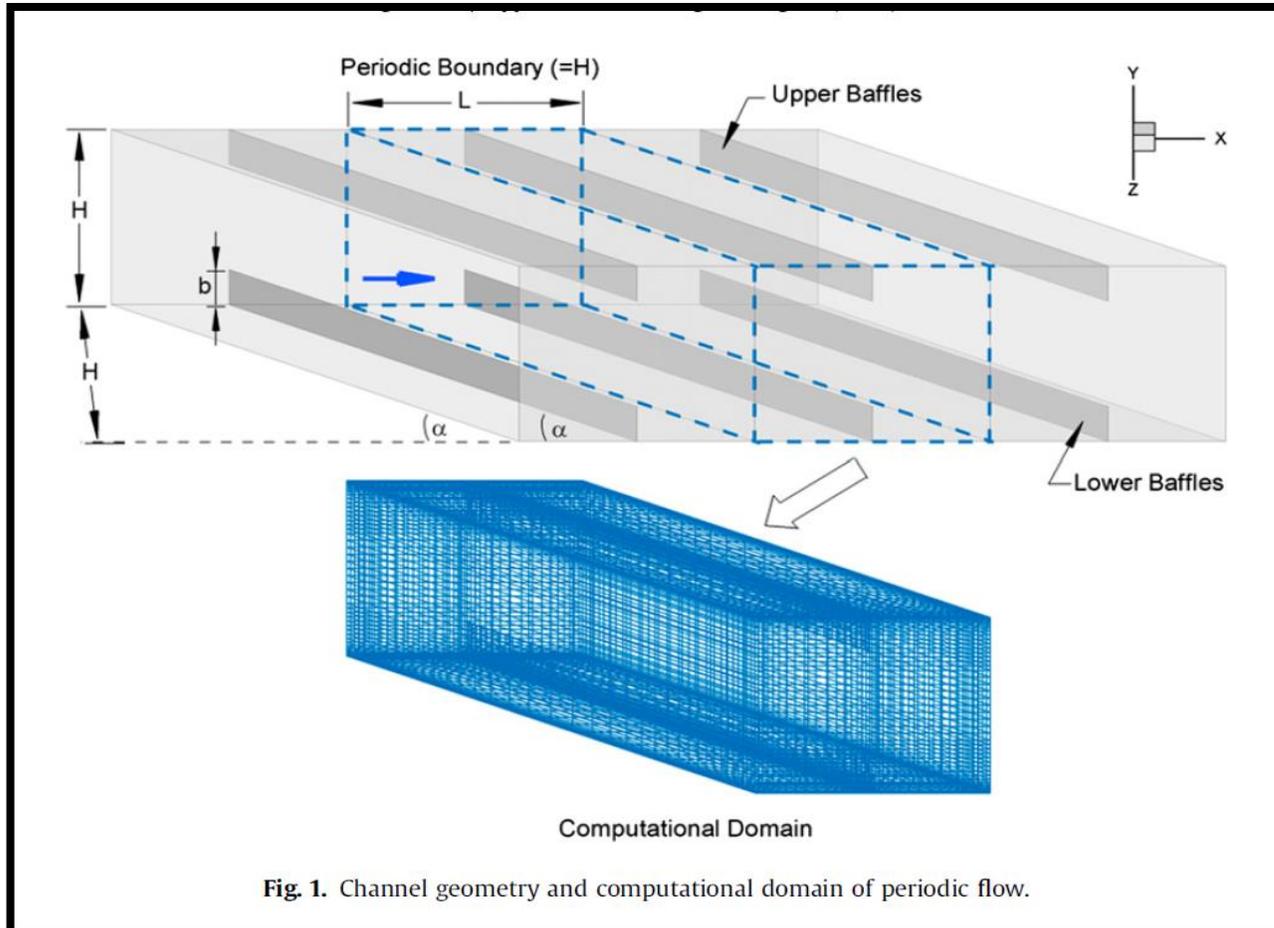
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Contents to be covered...

- Background
- Modelling and Methodology
- Results and Validation



Background



Objective

1. Effects of different geometry on flow behaviors in the channel.
2. Investigate heat transfer behaviors in a three-dimensional isothermal wall square channel fitted with 30-angled baffles.
3. Generate a pair of streamwise counter-rotating vortex (P-vortex) flows through the tested channel.

Figure 1: Streamline plot & Heat transfer behavior (right) and Channel Geometry (left)



Modelling And Methodology



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Approach

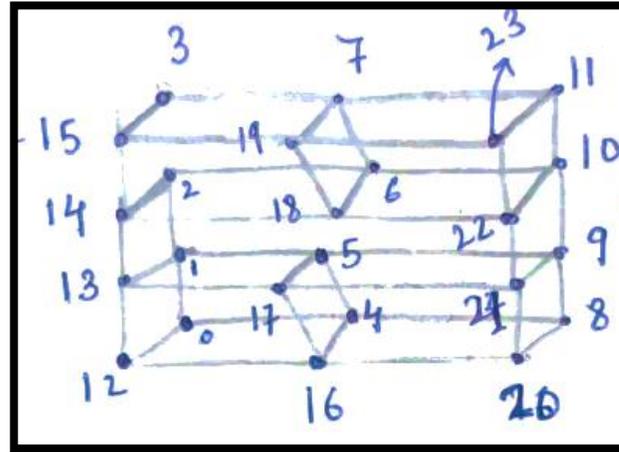
Multiblock
Meshing

Merging/
Baffle
Modelling

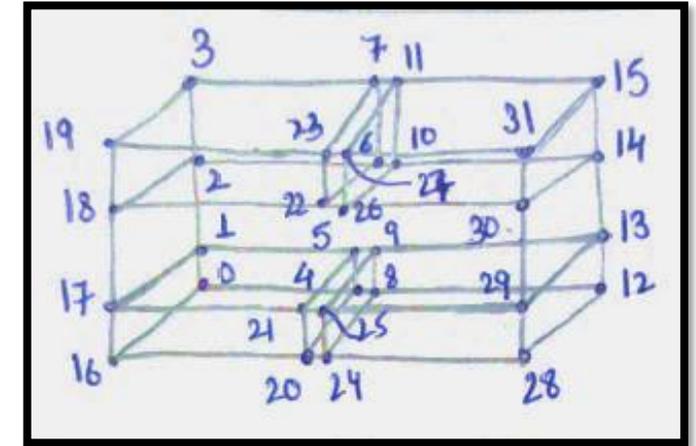
2D
Geometry
Modelling
with Baffles

3D
Geometry
Modeling
with Baffles

- ✓ Created multiple blocks in OpenFOAM.
- ✓ Merged and Meshed multi blocks.
- ✓ Created a single baffle with 2D multiblock.
- ✓ Created two 90 deg inline baffles with 2D multiblock.
- ✓ Created two 90 deg inline baffles with 3D multiblock.
- ✓ Created 30 deg line inline baffles with 3D block.



Method I



Method II

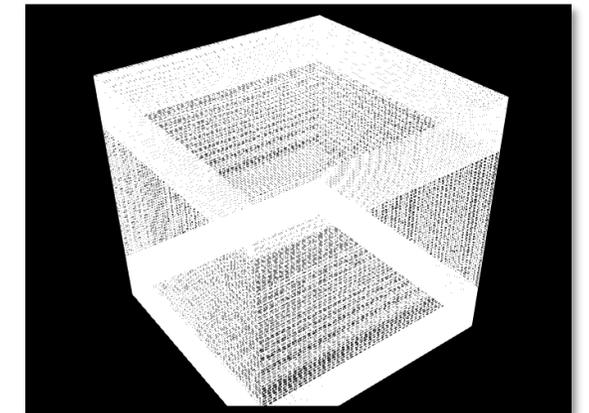
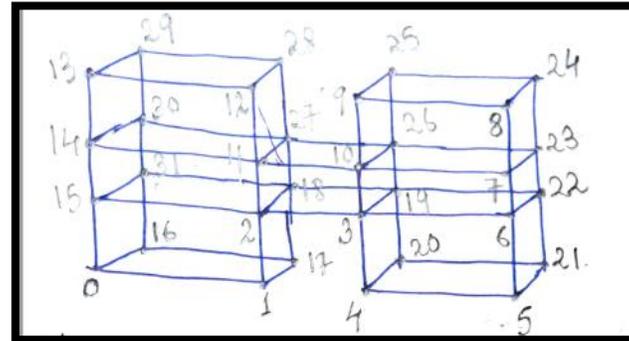


Figure 2: Square channel created by Multiblock Meshing and Merging Method.

Method III is most suitable for creating baffles without defining any additional source code.



Modeling and Meshing

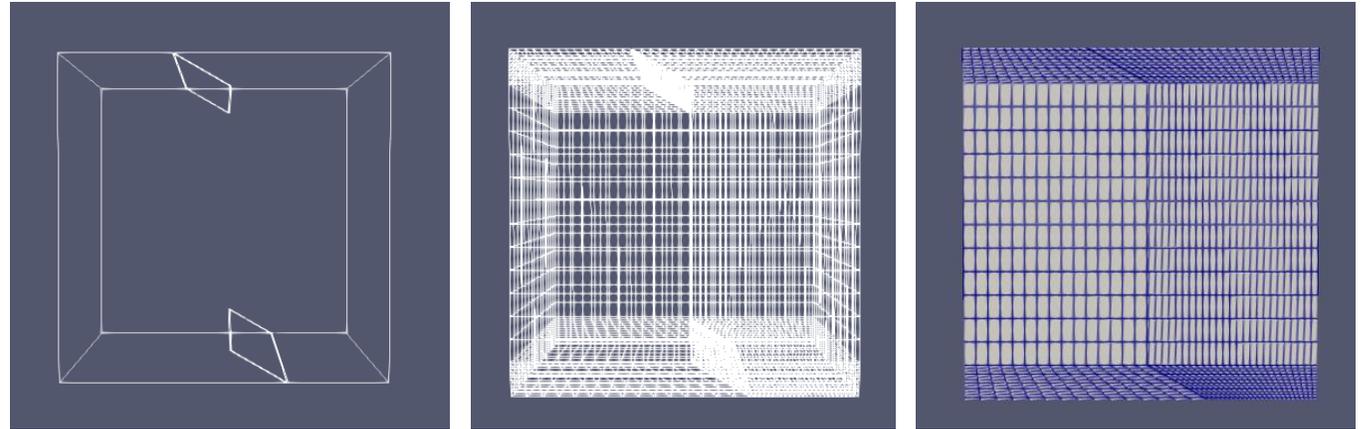
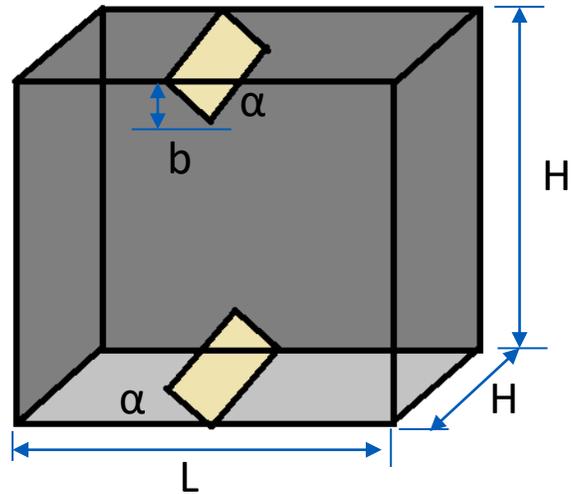


Figure 3: Square channel with 30 deg inline angled baffle turbulators with b/H ratio 0.1 for PR 1

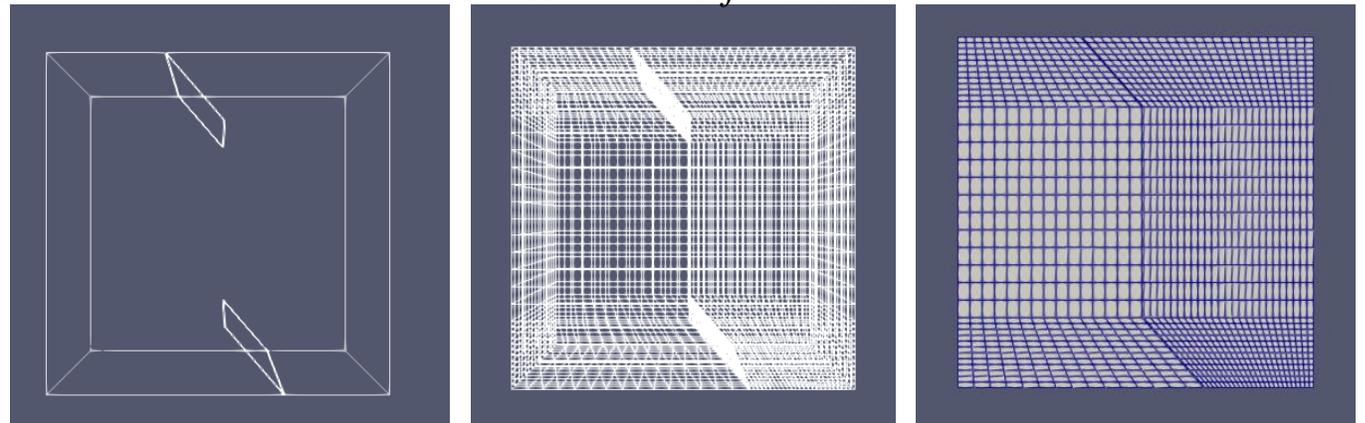


Figure 4: Square channel with 30 deg inline angled baffle turbulators with b/H ratio =0.2 for PR 1

H = Height of Channel = 0.05m
 L = Length of Channel = 0.05 (i.e., $L=H$)
Where, $L= H, 1.5H$ or $2H$.
Also, L/H = Pitch ratio (PR) = 1.0 ,1.5, 2.0
 b = Baffle Height
Where b/H = blockage ratio (BR) = 0.1,0.2,0.3.
 α = Baffle angle – 30 deg

Total Mesh Count is around ~96000



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Methodology

- ✓ Simulation for Flow through the channel was performed for checking the flow & heat transfer behavior for different geometry.
- ✓ Simulation running time – 1000 s with 0.001-time steps.
- ✓ *buoyantSimpleFoam* solver is used.
- ✓ Turbulence Property - Laminar
- ✓ Inlet Velocity – 0.316 m/s
- ✓ Density of fluid- 0.836 kg/m³
- ✓ Kinematic viscosity of fluid – 1.568 e-5m²/s

```
boundary
(
  outlet
  {
    type cyclic;
    neighbourPatch inlet;
    faces
    (
      (15 0 16 31)
      (14 15 31 30)
      (13 14 30 29)
    );
  }
  inlet
  {
    type cyclic;
    neighbourPatch outlet;
    faces
    (
      (5 6 22 21)
      (6 7 23 22)
      (7 8 24 23)
    );
  }
  walls
  {
    type wall;
    faces
    (
      (1 0 16 17)
      (2 1 17 18)
      (3 2 18 19)
      (4 3 19 20)
      (5 4 20 21)
      (9 8 24 25)
      (10 9 25 26)
      (11 10 26 27)
      (12 11 27 28)
      (13 12 28 29)
    );
  }
  frontBack
  {
    type empty;
    faces
    (
      (0 15 2 1)
      (15 14 11 2)
      (14 13 12 11)
      (2 11 10 3)
      (4 3 6 5)
      (3 10 7 6)
      (10 9 8 7)
      (16 17 18 31)
      (31 18 27 30)
      (30 27 28 29)
      (18 19 26 27)
      (20 21 22 19)
      (19 22 23 26)
      (26 23 24 25)
    );
  }
);
```

blockMeshDict File

```
dimensions [0 1 -1 0 0 0];
internalField uniform (0.316 0 0);
boundaryField
{
  #includeEtc "caseDicts/setConstraintTypes"
  walls
  {
    type noSlip;
  }
  inlet
  {
    type cyclic;
    value uniform (0.316 0 0);
  }
  outlet
  {
    type cyclic;
  }
  frontBack
  {
    type empty;
  }
}
```

U file



Thermophysical Properties & other parameters

```
dimensions      [1 -1 -2 0 0 0];
internalField   uniform 101325;
boundaryField
{
    #includeEtc "caseDicts/setConstraintTypes"

    walls
    {
        type          fixedFluxPressure;
        gradient      uniform 0;
        value         uniform 101325;
    }
    inlet
    {
        type          cyclic;
        gradient      uniform 0;
        value         uniform 101325;
    }
    outlet
    {
        type          cyclic;
        value         uniform 101325;
    }
    frontBack
    {
        type          empty;
    }
}
```

P_pgh file

```
internalField   uniform 300;
boundaryField
{
    walls
    {
        type          fixedValue;
        value         uniform 310;
    }
    inlet
    {
        type          cyclic;
        value         uniform 300;
    }
    outlet
    {
        type          cyclic;
        value         uniform 300;
    }
    frontBack
    {
        type          empty;
    }
}
```

T file

```
thermoType
{
    type          heRhoThermo;
    mixture       pureMixture;
    transport     const;
    thermo        hConst;
    equationOfState perfectGas;
    specie        specie;
    energy        sensibleEnthalpy;
}

mixture
{
    specie
    {
        molweight    28.96;
    }
    thermodynamics
    {
        Cp          1004.9;
        Hf          0;
    }
    transport
    {
        mu          1.846e-05;
        Pr          0.707;
    }
}
```

Thermo physical Properties file



Results and Validation



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Flow Behaviors in Channel

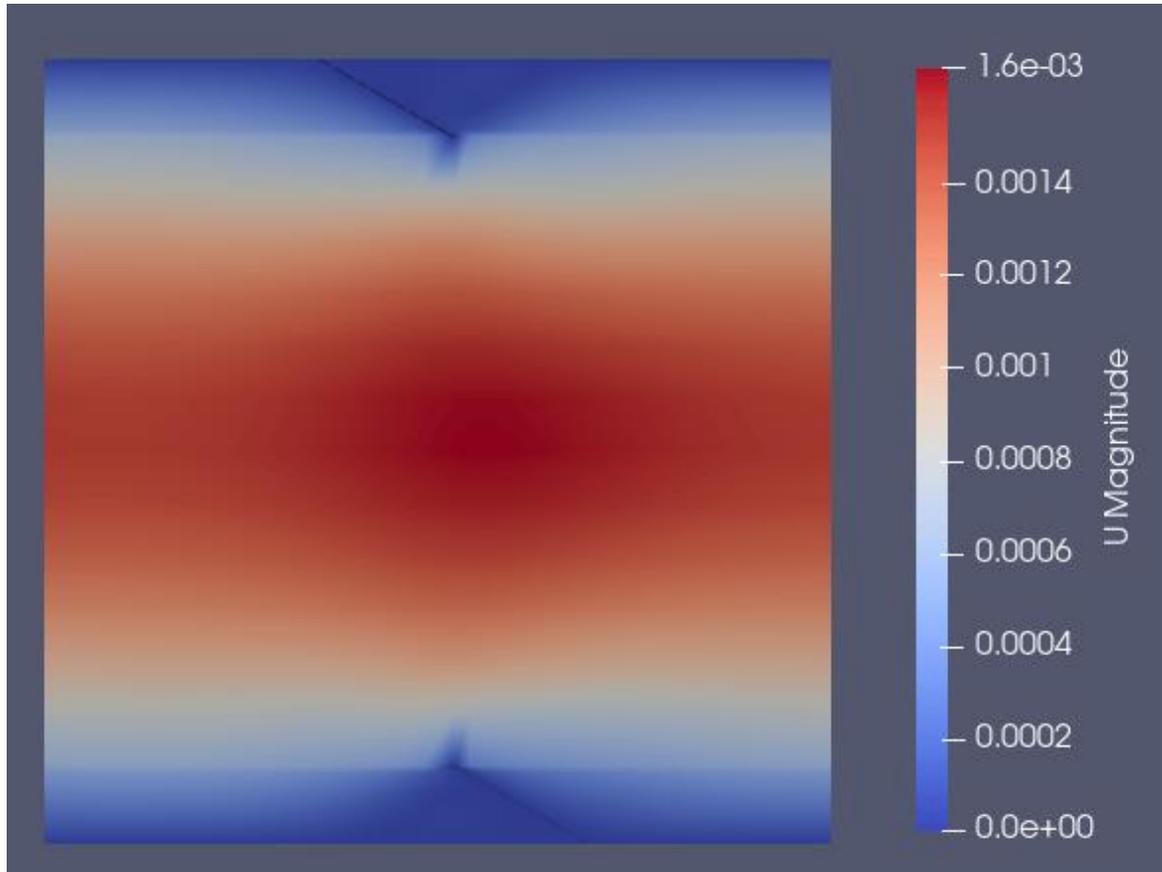


Figure 5: Velocity contour and velocity Profile plotted for 30 deg baffle with $b/H=0.1$

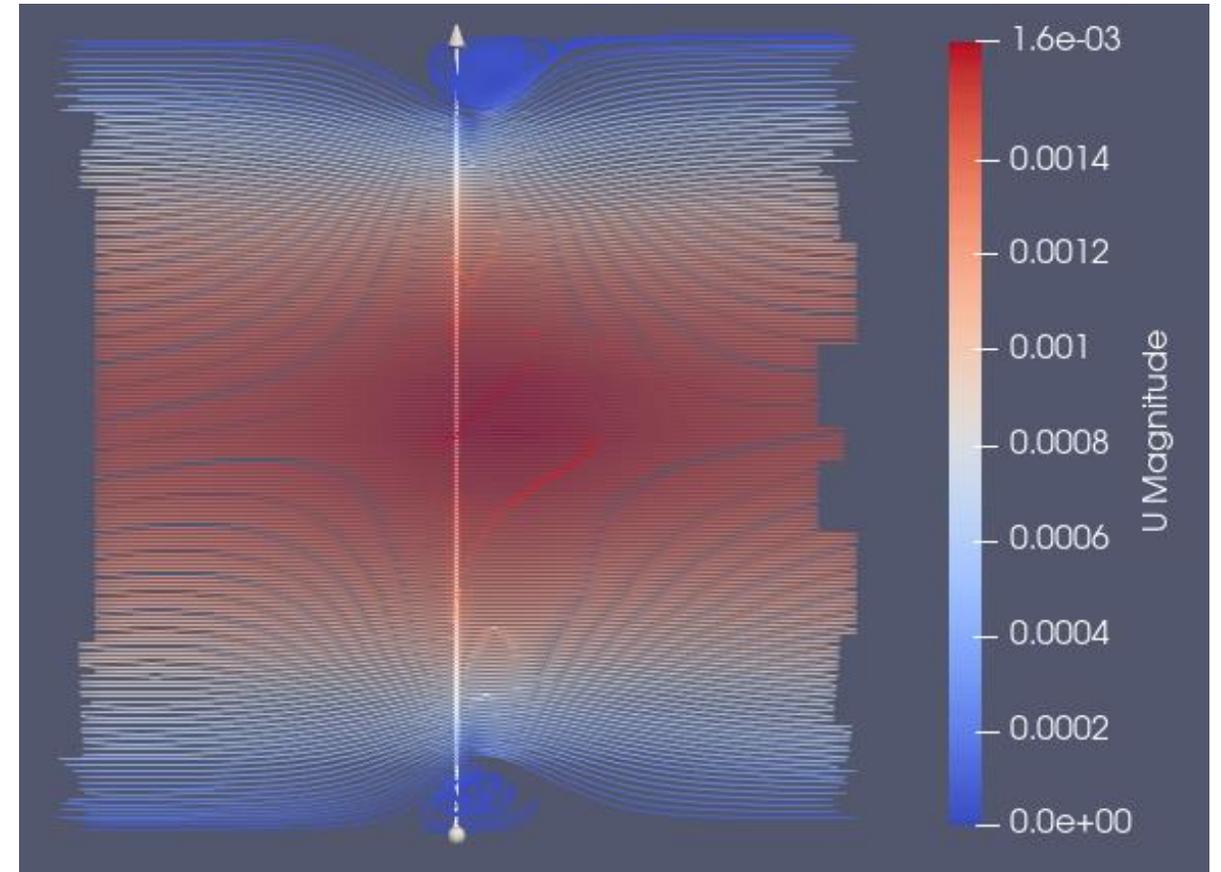


Figure 6: Streamline Contour for 30 deg baffle with $b/H=0.1$



Continue ...

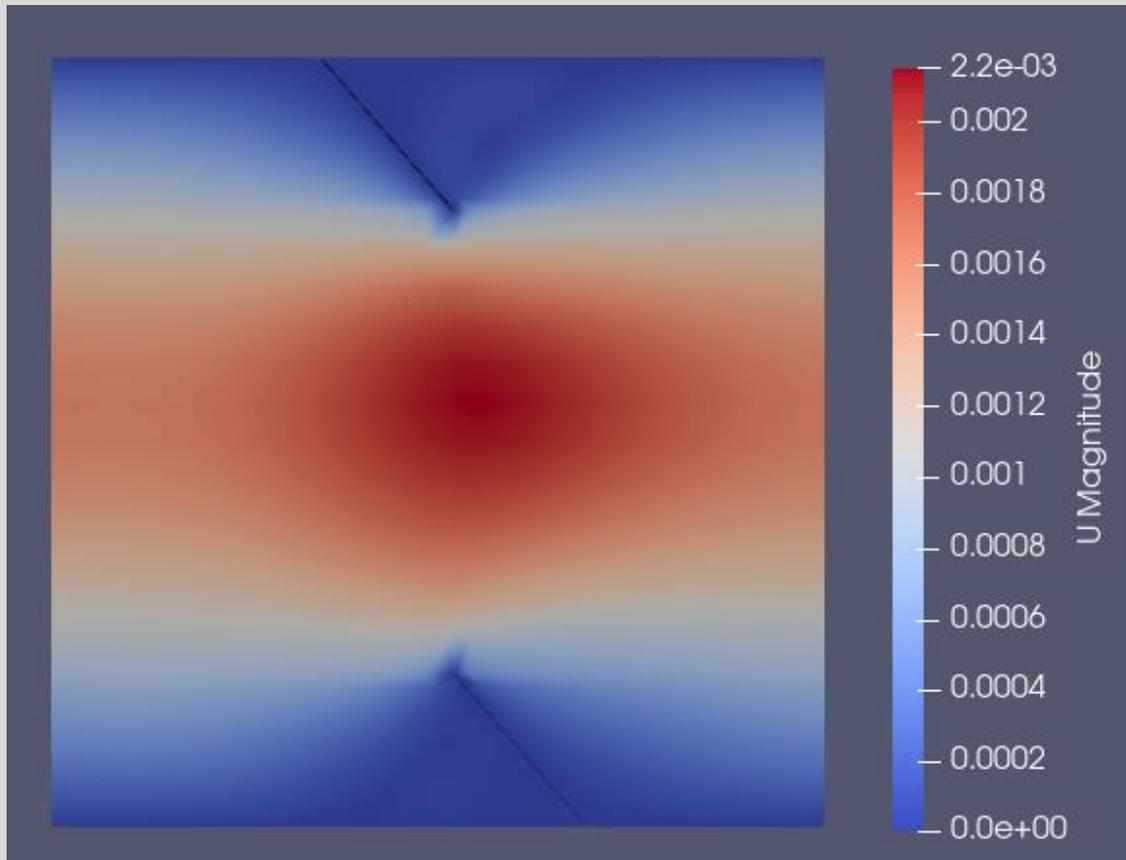


Figure 7: Velocity contour and velocity Profile plotted for 30 deg baffle with $b/H=0.2$

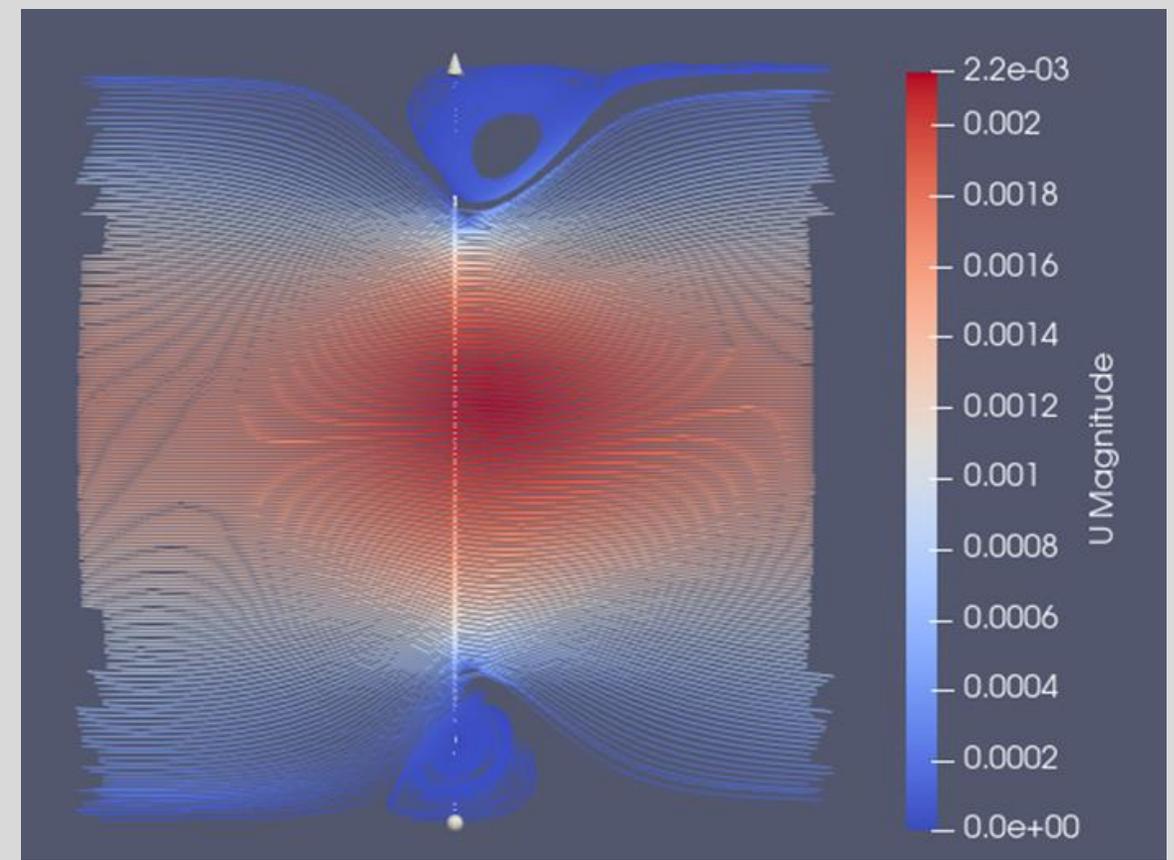
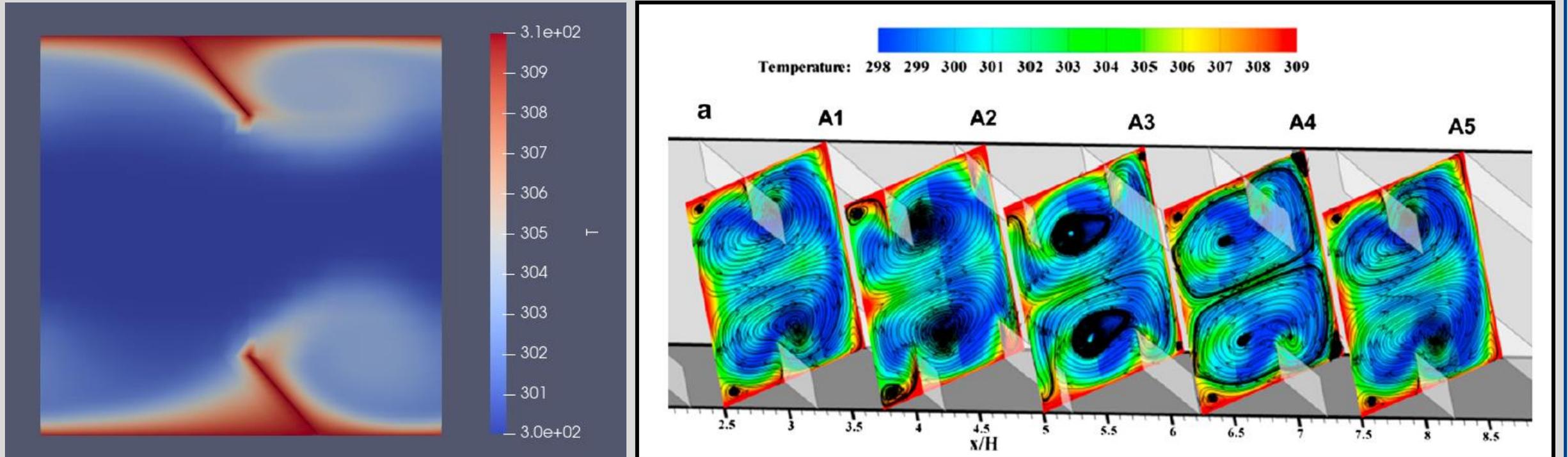


Figure 8: Streamline Contour for 30 deg baffle with $b/H=0.2$



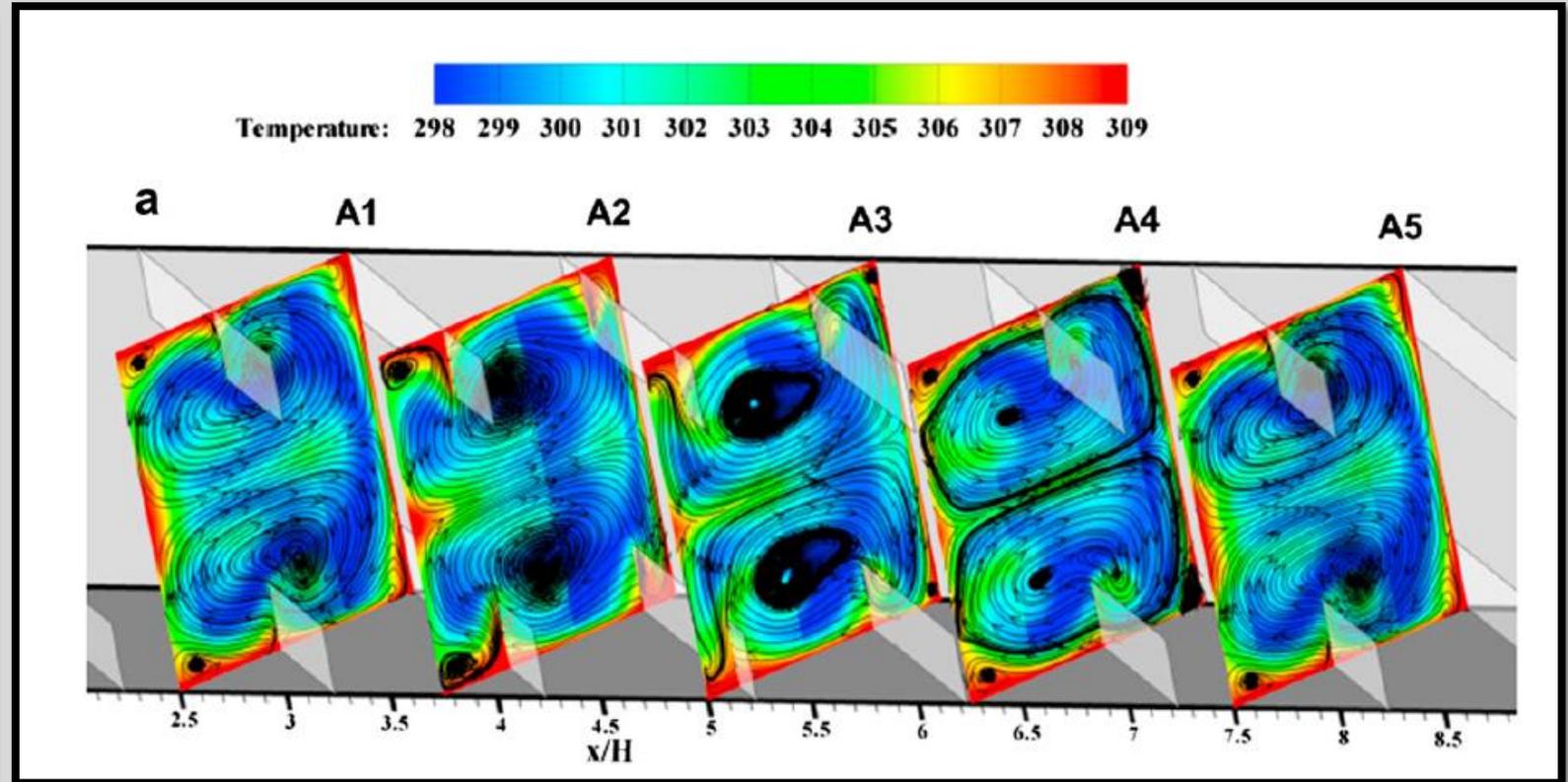
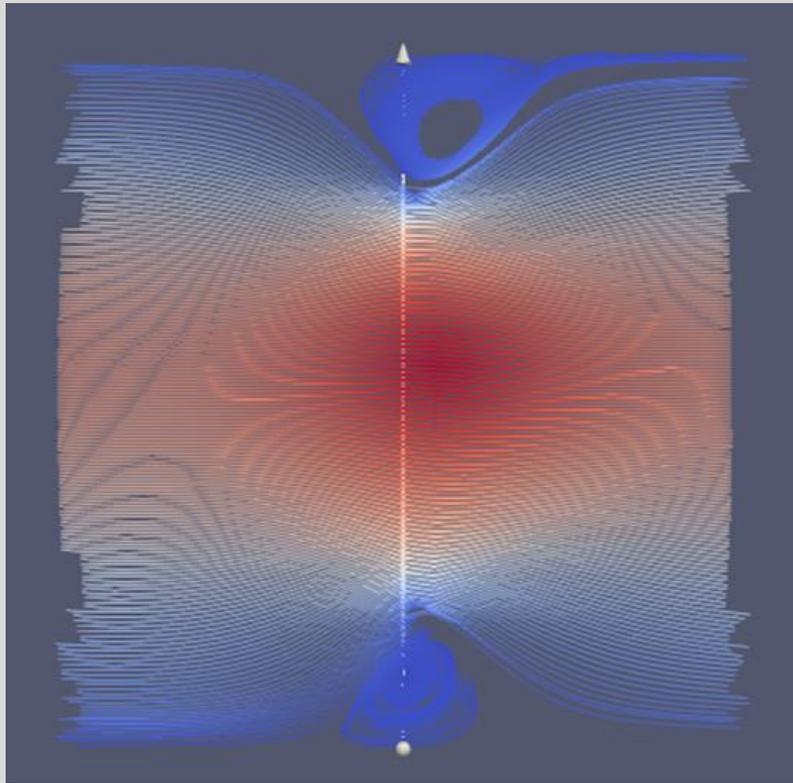
Heat Transfer behaviour in Channels



✓ Objective 2: Temperature plot almost like the reference paper with 0.6% to 1% deviation.



Pair of Vortex Formation in the Channel



✓ Objective 3 : Pair of streamwise counter-rotating vortex (P-vortex) flows through the channel can be seen.





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